

三肢剪力墙在水平荷载作用下内力位移解析解法

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摘要: 随着高层结构应用日益广泛, 多肢剪力墙及其组合结构成为高层结构中不可或缺的一部分, 但是在土木类规范、教材中对于多肢剪力墙内力求解的核心部分: 对连梁约束弯矩构成的微分方程组采取叠加分配的计算方法有失偏颇。文章对连梁约束弯矩求解过程中产生的 2 个二阶微分方程组采取转化为 4 个一阶微分方程组, 进而求微分方程组系数矩阵的基解矩阵的办法, 从而给出多肢剪力墙中三肢剪力墙在 3 种水平荷载作用下内力、位移的解析计算公式。比较两种方法的内力计算结果与 Midas 计算模型内力的差距, 发现运用传统方法连梁的内力结果较计算模型和文中计算小 10% 左右。该方法避免对连梁刚度以及连梁相对位置和墙肢整体性等一系列参数的讨论, 避开分配系数这个概念, 改进传统解析方法, 为剪力墙与框架组合结构内力、位移解析表达式的推导夯实基础。

关键词: 连梁; 三肢剪力墙; 内力; 位移; 解析解

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Analytical solution for internal force and displacement of three-limb shear wall under horizontal load

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Abstract: Multi-limb shear walls and the relevant composite structures are essential components of high-rise structures in modern designs. However, it is inaccurate to demonstrate the core method that calculating the internal force of multi-limb walls in codes, specifications, and textbooks of civil engineering, in which the superposition distribution method is adopted to solve the differential equations for the restrained moment of coupling beams. In this paper, a new method is proposed to solve the inertial forces and displacements of three limb shear walls under three types of horizontal loads, and the analytical equations are given. The method solves the restrained bending moment of coupling beams by transforming the two second-order differential equations into the four first-order differential equations. Then the fundamental solution matrix is obtained for the coefficient matrix of differential equations. By comparing the internal force calculated by these two methods with those from Midas, it is found that the results from the traditional method are 10% less than those from the FE method and the new method. Besides, the new method improves the traditional way by avoiding the concept of distribution factor, the stiffness and the relevant position of coupling beams, and the integrity of the shear wall, etc. The proposed new method can provide reference for the expression of internal forces and displacements in composite structures of shear wall and frame.

Keywords: coupling beam; three-limb shear wall; internal force; displacement; analytical solutions

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引言

高层建筑的日发展，钢筋混凝土剪力墙以其较大的抗侧移刚度，能有效控制结构水平位移，广泛应用于高层建筑结构中。对双肢（或多肢）剪力墙结构，内力位移计算尤为重要。包世华等^[1]总结出了多肢剪力墙的几种内力解法：连梁连续化的分析方法、带刚域框架的算法、有限单元和有限条带法，后两种适应于计算机运算。连梁连续化的分析方法适用于剪力墙内力位移解析公式的推导。在整体小开口剪力墙中，可按材料力学方法略加修正进行计算。双肢（或多肢）剪力墙一般是采用连续化方法，以沿竖向连续分布的连杆代替各层连梁的作用，用结构力学方法原理，以连梁跨中剪力为基本未知量，由切口处位移协调条件建立二阶常微分方程组。梁启智等^[2-4]用解微分方程组法解了多肢墙内力问题，并把此方法推广到空间剪力墙结构，计算机编程应用于高层结构较好地解决了内力问题，但其结果中设而不求的参数较多，方程并未求解，不利于手算。梁启智等^[5]、王慕勇等^[6]计算内力时简化了计算机编程，同样取得了较好结果。但都没有给出具体解析式，最后通过数值计算解决问题。

张铜生等^[7]在剪力墙结构计算存在的问题中提到，微分方程直接解答比较困难，一般采用以解析、半解析方法为基础的常微分方程求解器方法，如张铜生^[8]对不等肢双肢墙的计算等。本文目标是解决多肢剪力墙内力解析式的问题，即在一定假设前提下通过直接解微分方程组，解出多肢墙中三肢墙内力位移解析式。在包世华等^[1]提出的解法中，当解微分方程组计算多肢剪力墙的内力和位移时，是采取把 n 个微分方程叠加变为一个微分方程进行求解，然后按连梁的分配系数 η_i 进行分配；李爱群等^[9]对分配系数做了大量试验得出： η_i 与连梁的刚度和水平，竖直位置及整体参数 α 有关，其结果需要查表可得，致其使用范围受到限制，因此求出多肢剪力墙内力解析式是必要的。从数学角度来看，传统方法求出的解析式可信度不高，因为每一个约束弯矩之间并不独立，有其复杂的相关关系，这是由系数矩阵决定的，简单的微分方程的叠加求解微分方程导致产生的结果不知道与实际结果偏大还是偏小，可能导致其结果与实际相差甚远，随着建筑物高度的增加，内力计算上会产生比较大的误差，如

许铁生等^[10]对任意水平荷载作用下多肢框架剪力墙内力的求解，包世华^[11-12]对框肢剪力墙和落地剪力墙及考虑与壁式框架协同工作时内力位移计算会有累计误差。而对微分方程组的求解数学上是有明确解答的，王高熊等^[13]微分方程书中有严格证明。

本文通过对微分方程组直接求解，对三肢剪力墙的内力位移求解，作出解析解答，避免连梁分配系数这一概念，求出其内力位移解析式，对之后更多肢数的剪力墙的解析解答做出理论铺垫，拓展解析解答的使用范围，夯实这一理论基础。

1 连梁约束弯矩解析式的推导

1.1 计算模型的建立

根据基本假定^[1]：①将每一楼层处的连梁简化为均布在整个楼层高度上的连续连杆（见图 1）；②各墙肢在同一水平上侧向位移相等，且在同一标高处转角和曲率相等；③沿高度层高相近，刚度不变。由于假设连梁横向无限刚度，故水平无相对位移，只有在纵向有竖向位移，当截开连梁时，有以下 3 种相对竖向位移。

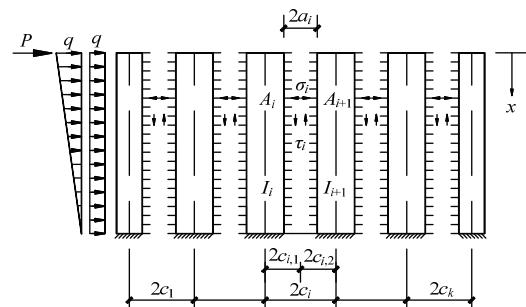


图 1 多肢墙的基本体系

Fig.1 The basic system of multi limb walls

(1) 由弯曲变形使切口处产生的相对位移：

$$\delta_{2i} = -2c_i \theta_m \quad (1)$$

(2) 由墙肢轴向变形产生的切口处相对位移：

$$\begin{aligned} \delta_{2i} &= \frac{1}{E} \left(\frac{1}{A_i} + \frac{1}{A_{i+1}} \right) \int_{x=0}^{H-x} \int \tau_i(x) dx dx - \\ &\quad \frac{1}{EA_i} \int_{x=0}^{H-x} \tau_{i-1}(x) dx dx - \frac{1}{EA_{i+1}} \int_{x=0}^{H-x} \tau_{i+1}(x) dx dx \quad (2) \end{aligned}$$

(3) 由于连梁弯曲和剪切变形产生的位移：

$$\delta_{3i}(x) = 2\tau_i(x) \left(\frac{a_i^3 h}{3EI_{bi}} + \frac{\mu a_i h}{GA_{bi}} \right) = \frac{2a_i^3 h}{3EI_{bi}} \tau_i(x) \quad (3)$$

$$\text{式中: } \tilde{I}_{bi} = \frac{I_{bi}}{1 + \frac{3\mu EI_{bi}}{a_i^2 GA_{bi}}}$$

第 i 连梁切口处的变形连续条件为:

$$\delta_i = \delta_{li}(x) + \delta_{2i}(x) + \delta_{3i}(x) = 0 \quad (4)$$

令 $m_i(x) = 2c_i \tau_i(x)$ 为第 i 列连梁的约束弯矩。将式(1)~式(4)叠加乘 $2c_i$ 微分两次, 可得到任意标高处连系梁中的位移协调方程为:

$$\begin{aligned} -4c_i^2 \theta_m'' - \frac{1}{E} \left(\frac{1}{A_i} + \frac{1}{A_{i+1}} \right) m_i(x) + \frac{c_i}{c_{i-1}} \frac{1}{EA_i} m_{i-1}(x) + \\ \frac{c_i}{c_{i+1}} \frac{1}{EA_{i+1}} m_{i+1}(x) + \frac{2a_i^3 h}{3E \tilde{I}_{bi}} m_i''(x) = 0, \quad i = 1, 2, \dots, k \end{aligned} \quad (5)$$

根据曲率与弯矩的关系, 求导可得:

$$\theta_m'' = -\frac{1}{E \sum I_i} (V_p(x) - m(x)) \quad (6)$$

取 $i=3$ 化简得:

$$\begin{cases} \frac{4c_1^2}{E(I_1 + I_2 + I_3)} [V_p(x) - (m_1(x) + m_2(x))] - \\ \frac{1}{E} \left(\frac{1}{A_1} + \frac{1}{A_2} \right) m_1(x) + \frac{c_1}{c_2} \frac{1}{EA_2} m_2(x) + \frac{2a_1^3 h}{3E \tilde{I}_{bl}} m_1''(x) = 0 \\ \frac{4c_2^2}{E(I_1 + I_2 + I_3)} [V_p(x) - (m_1(x) + m_2(x))] - \\ \frac{1}{E} \left(\frac{1}{A_2} + \frac{1}{A_3} \right) m_2(x) + \frac{c_2}{c_1} \frac{1}{EA_1} m_1(x) + \frac{2a_2^3 h}{3E \tilde{I}_{b2}} m_2''(x) = 0 \end{cases} \quad (7)$$

化简(7)可得连梁约束弯矩微分方程式:

$$\begin{aligned} m_1''(x) &= (\alpha_1 + \beta_1)m_1(x) + (\alpha_1 - \gamma_1)m_2(x) - \alpha_1 V_p(x) \\ m_2''(x) &= (\alpha_2 + \beta_2)m_2(x) + (\alpha_2 - \gamma_2)m_1(x) - \alpha_2 V_p(x) \end{aligned} \quad (8)$$

令相关参数如下:

$$\begin{aligned} \alpha_1 &= \frac{6c_1^2 \tilde{I}_{bl}}{a_1^3 (I_1 + I_2 + I_3) h}; \quad \alpha_2 = \frac{6c_2^2 \tilde{I}_{b2}}{a_2^3 (I_1 + I_2 + I_3) h} \\ \beta_1 &= \frac{3\tilde{I}_{bl}}{2a_1^3 h} \frac{A_1 + A_2}{A_1 A_2}; \quad \beta_2 = \frac{3\tilde{I}_{b2}}{2a_2^3 h} \frac{A_2 + A_3}{A_2 A_3} \\ \gamma_1 &= \frac{3c_1 \tilde{I}_{bl}}{2a_1^3 c_2 A_2 h}; \quad \gamma_2 = \frac{3c_2 \tilde{I}_{b2}}{2a_2^3 c_1 A_1 h} \end{aligned} \quad (9)$$

式中: $2a_i$ 为第 i 跨连梁的计算跨度; $2c_i$ 为第 i 跨墙肢轴线间距; θ_m 为墙肢弯曲变形产生的转角; m_i 为第 i 个连梁的约束弯矩; V_p 为外荷载对 x 截面的总剪力; I_i 为第 i 列墙肢的截面惯性矩; \tilde{I}_{bi} 为第 i 个连梁的折算惯性矩。

1.2 微分方程的求解

1.2.1 基本理论

依据文献[2]解微分方程组的思路进行求解。把式(8)2个二阶微分方程转化为4个一阶微分方程进行求解, 化为 $\dot{x} = Ax + \beta$ 的形式, 先弃去方程非齐次项, 先研究 $\dot{x} = Ax$ 的形式的通解, 求出该微分方程组的基解矩阵 $\Phi(x)$ 即为微分方程通解, 由基解矩阵 $\Phi(x)$ 与矩阵 A 的特征值与特征向量 φ_i 的关系可得: $\Phi(x) = [e^{\lambda_1 x} \varphi_1, e^{\lambda_2 x} \varphi_2, \dots, e^{\lambda_n x} \varphi_n]$ 可知, 求出矩阵 A 的特征值、特征向量, 即可求得基解矩阵 $\Phi(x)$, 然后结合常数变异法找满足含非齐次项 β 向量的微分方程组的特解即可。

1.2.2 基解矩阵的求解

不妨设 $y_1 = m_1(x)$, $y_2 = m_2(x)$, $y_3 = m_1'(x)$, $y_4 = m_2'(x)$, 由式(9)可得方程:

$$\begin{cases} \dot{y}_1 = y_3 \\ \dot{y}_2 = y_4 \\ \dot{y}_3 = (\alpha_1 + \beta_1)y_1 + (\alpha_1 - \gamma_1)y_2 - \alpha_1 V_p(x) \\ \dot{y}_4 = (\alpha_2 - \gamma_2)y_1 + (\alpha_2 + \beta_2)y_2 - \alpha_2 V_p(x) \end{cases} \quad (10)$$

将式(11)写成矩阵形式为:

$$\dot{\mathbf{y}}' = A\mathbf{y} + \beta; \quad \mathbf{y} = [y_1 \quad y_2 \quad y_3 \quad y_4]^T$$

该方程的特征矩阵和荷载列向量为:

$$A = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ \alpha_1 + \beta_1 & \alpha_1 - \gamma_1 & 0 & 0 \\ \alpha_2 - \gamma_2 & \alpha_2 + \beta_2 & 0 & 0 \end{bmatrix}; \quad \beta = \begin{bmatrix} 0 \\ 0 \\ -\alpha_1 V_p(x) \\ -\alpha_2 V_p(x) \end{bmatrix}$$

先弃去方程非齐次项, 研究 $\dot{x} = Ax$ 的形式的通解。设 $\psi(t) = \varphi_i e^{\lambda t}$, $c \neq 0$ 的解, 其中常数 λ 和向量 φ_i 是待定的, 将 $\psi(t)$ 带入 $\lambda e^{\lambda t} \varphi_i = A e^{\lambda t} \varphi_i$, 因为 $e^{\lambda t} \neq 0$, 上式变为方程 $(\lambda E - A)\varphi_i = 0$, 这就表示 $\varphi_i e^{\lambda t}$ 是解的充要条件就是常数 λ 和向量 φ_i 满足方程, 方程可以看作是向量 φ_i 的 n 个分量的一个齐次线性代数方程组, 根据线性代数知识, 这个方程具有非零解的充要条件的就是 λ 满足方程

$$\det(\lambda E - A) = 0 \quad \text{即:}$$

$$\begin{bmatrix} \lambda & 0 & -1 & 0 \\ 0 & \lambda & 0 & -1 \\ -(\alpha_1 + \beta_1) & -(\alpha_1 - \gamma_1) & \lambda & 0 \\ -(\alpha_2 - \gamma_2) & -(\alpha_2 + \beta_2) & 0 & \lambda \end{bmatrix} = 0$$

$$\text{令: } \delta = (\alpha_1 + \beta_1)(\alpha_2 + \beta_2) - (\alpha_1 - \gamma_1)(\alpha_2 - \gamma_2)$$

$$\mu = \alpha_1 + \beta_1 + \alpha_2 + \beta_2 \quad (11)$$

由式(11)可得: $\lambda^4 - \lambda^2\mu + \delta = 0$ (12)

由式(12)可解得:

$$\lambda_1 = \frac{\sqrt{2\mu+2\sqrt{\mu^2-4\delta}}}{2} = k_1$$

$$\lambda_2 = -\frac{\sqrt{2\mu+2\sqrt{\mu^2-4\delta}}}{2} = -k_1$$

$$\lambda_3 = \frac{\sqrt{2\mu-2\sqrt{\mu^2-4\delta}}}{2} = k_2$$

$$\lambda_4 = -\frac{\sqrt{2\mu-2\sqrt{\mu^2-4\delta}}}{2} = -k_2$$

将式(12)求出的解代入矩阵 $\lambda E - A$ 求出特征向量。

当 $\lambda_1 = k_1$ 时特征向量为:

$$\varphi_1 = C_1 \begin{pmatrix} \frac{1}{k_1} & \frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1(\alpha_1 - \gamma_1)} & 1 & \frac{k_1^2 - (\alpha_1 + \beta_1)}{\alpha_1 - \gamma_1} \end{pmatrix}^T$$

当 $\lambda_2 = -k_1$ 时特征向量为:

$$\varphi_2 = C_2 \begin{pmatrix} -\frac{1}{k_1} & -\frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1(\alpha_1 - \gamma_1)} & 1 & \frac{k_1^2 - (\alpha_1 + \beta_1)}{\alpha_1 - \gamma_1} \end{pmatrix}^T$$

当 $\lambda_3 = k_2$ 时特征向量为:

$$\varphi_3 = C_3 \begin{pmatrix} \frac{1}{k_2} & \frac{[k_2^2 - (\alpha_1 + \beta_1)]}{k_2(\alpha_1 - \gamma_1)} & 1 & \frac{k_2^2 - (\alpha_1 + \beta_1)}{\alpha_1 - \gamma_1} \end{pmatrix}^T$$

当 $\lambda_4 = -k_2$ 时特征向量为:

$$\varphi_4 = C_4 \begin{pmatrix} -\frac{1}{k_2} & -\frac{[k_2^2 - (\alpha_1 + \beta_1)]}{k_2(\alpha_1 - \gamma_1)} & 1 & \frac{k_2^2 - (\alpha_1 + \beta_1)}{\alpha_1 - \gamma_1} \end{pmatrix}^T \quad (13)$$

将式(13)的特征向量与对应的 $e^{\lambda t}$ 组合即可得到基解矩阵 $\Phi(x)$, 故通解为:

$$\left\{ \begin{array}{l} \text{当 } V_p = V_0 \text{ 时:} \\ \left\{ \begin{array}{l} y_1^* = \frac{V_0}{\delta} [\alpha_1(\alpha_2 + \beta_2) - \alpha_2(\alpha_1 - \gamma_1)] \\ y_2^* = \frac{V_0}{\delta} \left\{ \frac{\alpha_1[\delta - (\alpha_1 + \beta_1)\mu + (\alpha_1 + \beta_1)^2]}{(\alpha_1 - \gamma_1)} + \alpha_2(\alpha_1 + \beta_1) \right\} \end{array} \right. \\ \text{当 } V_p = \frac{V_0 x}{H} \text{ 时:} \\ \left\{ \begin{array}{l} y_1^* = \frac{V_0 x [\alpha_1(\alpha_2 + \beta_2) - \alpha_2(\alpha_1 - \gamma_1)]}{H\delta} \\ y_2^* = \frac{V_0 x}{H\delta} \left\{ \frac{\alpha_1[\delta - (\alpha_1 + \beta_1)\mu + (\alpha_1 + \beta_1)^2]}{(\alpha_1 - \gamma_1)} + \alpha_2(\alpha_1 + \beta_1) \right\} \end{array} \right. \\ \text{当 } V_p = V_0 \left[1 - \left(1 - \frac{x}{H}\right)^2 \right] \text{ 时:} \\ \left\{ \begin{array}{l} y_1^* = V_0 \left\{ \frac{\alpha_1[\mu - (\alpha_1 + \beta_1)] - \alpha_2(\alpha_1 - \gamma_1)}{\delta} \left(\frac{2x}{H} - \frac{x^2}{H^2} \right) - \frac{2\alpha_1[\mu^2 - (\alpha_1 + \beta_1)\mu - \delta] - 2\alpha_2(\alpha_1 - \gamma_1)\mu}{\delta^2 H^2} \right\} \\ y_2^* = V_0 \left\{ \frac{\alpha_1[\delta - (\alpha_1 + \beta_1)\mu + (\alpha_1 + \beta_1)^2] + \alpha_2(\alpha_1 - \gamma_1)(\alpha_1 + \beta_1)}{\delta(\alpha_1 - \gamma_1)} \left(\frac{2x}{H} - \frac{x^2}{H^2} \right) - \frac{2\alpha_1[\delta\mu - (\alpha_1 + \beta_1)\mu^2 + (\alpha_1 + \beta_1)^2\mu] - 2\alpha_2(\alpha_1 - \gamma_1)[\delta - (\alpha_1 + \beta_1)\mu]}{\delta^2 H^2(\alpha_1 - \gamma_1)} \right\} \end{array} \right. \end{array} \right. \quad (16)$$

$$\left\{ \begin{array}{l} y_1 = \frac{C_1}{k_1} e^{k_1 x} - \frac{C_2}{k_1} e^{-k_1 x} + \frac{C_3}{k_2} e^{k_2 x} - \frac{C_4}{k_2} e^{-k_2 x} \\ y_2 = C_1 \frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1(\alpha_1 - \gamma_1)} e^{k_1 x} - C_2 \frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1(\alpha_1 - \gamma_1)} e^{-k_1 x} + \\ C_3 \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2(\alpha_1 - \gamma_1)} e^{k_2 x} - C_4 \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2(\alpha_1 - \gamma_1)} e^{-k_2 x} \\ y_3 = C_1 e^{k_1 x} + C_2 e^{-k_1 x} + C_3 e^{k_2 x} + C_4 e^{-k_2 x} \\ y_4 = \frac{C_1 [k_1^2 - (\alpha_1 + \beta_1)]}{(\alpha_1 - \gamma_1)} e^{k_1 x} + C_2 \frac{k_1^2 - (\alpha_1 + \beta_1)}{(\alpha_1 - \gamma_1)} e^{-k_1 x} + \\ C_3 \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2(\alpha_1 - \gamma_1)} e^{k_2 x} + C_4 \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2(\alpha_1 - \gamma_1)} e^{-k_2 x} \end{array} \right. \quad (14)$$

式中: y_1 、 y_2 即为两连梁约束弯矩的通解; y_3 、 y_4 即为其导数。

1.2.3 常数变异法求特解

由于方程解的构成为齐次方程通解加非齐次方程特解, 现求其非齐次方程特解, 采用常数变异法, 将自由常数看作函数, 代入式(14)令其等于 β 向量, 用克拉默法则可解得:

$$\left\{ \begin{array}{l} C_1(x) = V_p(x) e^{-k_1 x} \frac{\alpha_1[k_2^2 - (\alpha_1 + \beta_1)] - \alpha_2(\alpha_1 - \gamma_1)}{2(k_1^2 - k_2^2)} \\ C_2(x) = V_p(x) e^{k_1 x} \frac{\alpha_1[k_2^2 - (\alpha_1 + \beta_1)] - \alpha_2(\alpha_1 - \gamma_1)}{2(k_1^2 - k_2^2)} \\ C_3(x) = -V_p(x) e^{-k_2 x} \frac{\alpha_1[k_1^2 - (\alpha_1 + \beta_1)] - \alpha_2(\alpha_1 - \gamma_1)}{2(k_1^2 - k_2^2)} \\ C_4(x) = -V_p(x) e^{k_2 x} \frac{\alpha_1[k_1^2 - (\alpha_1 + \beta_1)] - \alpha_2(\alpha_1 - \gamma_1)}{2(k_1^2 - k_2^2)} \end{array} \right. \quad (15)$$

求其原函数并代入式(15), 方程 y_1 、 y_2 中可得特解 y_1^* 、 y_2^* , 不同荷载对应不同特解得到式(10):

将式(16)特解与式(14)通解组合即为方程的解:

$$\begin{cases} y_1 = \frac{C_1}{k_1} e^{k_1 x} - \frac{C_2}{k_1} e^{-k_1 x} + \frac{C_3}{k_2} e^{k_2 x} - \frac{C_4}{k_2} e^{-k_2 x} + y_1^* \\ y_2 = \frac{C_1 [k_1^2 - (\alpha_1 + \beta_1)]}{k_1(\alpha_1 - \gamma_1)} e^{k_1 x} - C_2 \frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1(\alpha_1 - \gamma_1)} e^{-k_1 x} + \\ C_3 \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2(\alpha_1 - \gamma_1)} e^{k_2 x} - C_4 \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2(\alpha_1 - \gamma_1)} e^{-k_2 x} + y_2^* \end{cases} \quad (17)$$

$$\begin{aligned} y_1 &= \left\{ \frac{b_1 [k_2^2 - (\alpha_1 + \beta_1)] - b_2 (\alpha_1 - \gamma_1)}{(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} \right\} H(e^{k_1 x} + e^{-k_1 x}) - \left\{ \frac{b_1 [k_1^2 - (\alpha_1 + \beta_1)] - b_2 (\alpha_1 - \gamma_1)}{(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} \right\} H(e^{k_2 x} + e^{-k_2 x}) + y_1^* \\ y_2 &= \left\{ \frac{\left[b_1 \frac{k_2^2 - (\alpha_1 + \beta_1)}{\alpha_1 - \gamma_1} - b_2 \right] [k_1^2 - (\alpha_1 + \beta_1)]}{(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} \right\} H(e^{k_1 x} + e^{-k_1 x}) - \left\{ \frac{\left[b_1 \frac{k_1^2 - (\alpha_1 + \beta_1)}{\alpha_1 - \gamma_1} - b_2 \right] [k_2^2 - (\alpha_1 + \beta_1)]}{(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} \right\} H(e^{k_2 x} + e^{-k_2 x}) + y_2^* \end{aligned} \quad (18)$$

当荷载为均布荷载连梁约束弯矩为:

$$\begin{aligned} y_1 &= \left\{ \frac{b_1 [k_2^2 - (\alpha_1 + \beta_1)] - b_2 (\alpha_1 - \gamma_1)}{(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} \right\} \left[e^{k_1 x} \left(\frac{e^{-k_1 H}}{k_1} + H \right) - e^{-k_1 x} \left(\frac{e^{k_1 H}}{k_1} - H \right) \right] - \\ &\quad \left\{ \frac{b_1 [k_1^2 - (\alpha_1 + \beta_1)] - b_2 (\alpha_1 - \gamma_1)}{(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} \right\} \left[e^{k_2 x} \left(\frac{e^{-k_2 H}}{k_2} + H \right) - e^{-k_2 x} \left(\frac{e^{k_2 H}}{k_2} - H \right) \right] + y_1^* \\ y_2 &= \left\{ \frac{\left[b_1 \frac{k_2^2 - (\alpha_1 + \beta_1)}{\alpha_1 - \gamma_1} - b_2 \right] [k_1^2 - (\alpha_1 + \beta_1)]}{(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} \right\} \left[e^{k_1 x} \left(\frac{e^{-k_1 H}}{k_1} + H \right) - e^{-k_1 x} \left(\frac{e^{k_1 H}}{k_1} - H \right) \right] - \\ &\quad \left\{ \frac{\left[b_1 \frac{k_1^2 - (\alpha_1 + \beta_1)}{\alpha_1 - \gamma_1} - b_2 \right] [k_2^2 - (\alpha_1 + \beta_1)]}{(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} \right\} \left[e^{k_2 x} \left(\frac{e^{-k_2 H}}{k_2} + H \right) - e^{-k_2 x} \left(\frac{e^{k_2 H}}{k_2} - H \right) \right] + y_2^* \end{aligned} \quad (19)$$

当荷载为倒三角荷载连梁约束弯矩为:

$$\begin{aligned} y_1 &= \left\{ \frac{e^{-k_1 H} \{x_1 [k_2^2 - (\alpha_1 + \beta_1)] - x_2 (\alpha_1 - \gamma_1)\}}{k_1(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} + \frac{\{x_3 [k_2^2 - (\alpha_1 + \beta_1)] - x_4 (\alpha_1 - \gamma_1)\}}{(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} \right\} e^{k_1 x} - \left\{ \frac{e^{k_1 H} \{x_1 [k_2^2 - (\alpha_1 + \beta_1)] - x_2 (\alpha_1 - \gamma_1)\}}{k_1(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} \right. \\ &\quad \left. - \frac{\{x_3 [k_2^2 - (\alpha_1 + \beta_1)] - x_4 (\alpha_1 - \gamma_1)\}}{(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} \right\} e^{-k_1 x} - \left\{ \frac{e^{-k_2 H} \{x_1 [k_1^2 - (\alpha_1 + \beta_1)] - x_2 (\alpha_1 - \gamma_1)\}}{k_2(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} + \frac{\{x_3 [k_1^2 - (\alpha_1 + \beta_1)] - x_4 (\alpha_1 - \gamma_1)\}}{(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} \right\} e^{k_2 x} + \\ &\quad \left\{ \frac{e^{k_2 H} \{x_1 [k_1^2 - (\alpha_1 + \beta_1)] - x_2 (\alpha_1 - \gamma_1)\}}{k_2(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} - \frac{\{x_3 [k_1^2 - (\alpha_1 + \beta_1)] - x_4 (\alpha_1 - \gamma_1)\}}{(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} \right\} e^{-k_2 x} + y_1^* \\ y_2 &= \left\{ \frac{e^{-k_1 H} \{x_1 [k_2^2 - (\alpha_1 + \beta_1)] - x_2 (\alpha_1 - \gamma_1)\}}{k_1(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})(\alpha_1 - \gamma_1)} + \frac{\{x_3 [k_2^2 - (\alpha_1 + \beta_1)] - x_4 (\alpha_1 - \gamma_1)\}}{(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})(\alpha_1 - \gamma_1)} \right\} [k_1^2 - (\alpha_1 + \beta_1)] e^{k_1 x} - \\ &\quad \left\{ \frac{e^{k_1 H} \{x_1 [k_2^2 - (\alpha_1 + \beta_1)] - x_2 (\alpha_1 - \gamma_1)\}}{k_1(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})(\alpha_1 - \gamma_1)} - \frac{\{x_3 [k_2^2 - (\alpha_1 + \beta_1)] - x_4 (\alpha_1 - \gamma_1)\}}{(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})(\alpha_1 - \gamma_1)} \right\} [k_1^2 - (\alpha_1 + \beta_1)] e^{-k_1 x} - \\ &\quad \left\{ \frac{e^{-k_2 H} \{x_1 [k_1^2 - (\alpha_1 + \beta_1)] - x_2 (\alpha_1 - \gamma_1)\}}{k_2(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})(\alpha_1 - \gamma_1)} + \frac{\{x_3 [k_1^2 - (\alpha_1 + \beta_1)] - x_4 (\alpha_1 - \gamma_1)\}}{(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})(\alpha_1 - \gamma_1)} \right\} [k_2^2 - (\alpha_1 + \beta_1)] e^{k_2 x} + \\ &\quad \left\{ \frac{e^{k_2 H} \{x_1 [k_1^2 - (\alpha_1 + \beta_1)] - x_2 (\alpha_1 - \gamma_1)\}}{k_2(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})(\alpha_1 - \gamma_1)} - \frac{\{x_3 [k_1^2 - (\alpha_1 + \beta_1)] - x_4 (\alpha_1 - \gamma_1)\}}{(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})(\alpha_1 - \gamma_1)} \right\} [k_2^2 - (\alpha_1 + \beta_1)] e^{-k_2 x} + y_2^* \end{aligned} \quad (20)$$

1.2.4 确定边界条件

式(17)中 C_1, C_2, C_3, C_4 为任意常数, 由边界条件确定, 由边界条件得: 当 $x=0$ 时, 墙顶弯矩为 0; 当 $x=H$ 时, 墙底弯曲变形转角为 0 得:

$$y'(0)=0; \quad y(H)=0$$

将边界条件代入式(17)运用克拉默法则求出 C_1, C_2, C_3, C_4 , 解得:

当荷载为集中荷载连梁约束弯矩为:

令相关参数:

$$\begin{aligned}
 b_1 &= \frac{V_0}{\delta H} [\alpha_1(\alpha_2 + \beta_2) - \alpha_2(\alpha_1 - \gamma_1)] \\
 b_2 &= \frac{V_0}{\delta H} \left\{ \frac{\alpha_1[\delta - (\alpha_1 + \beta_1)\mu + (\alpha_1 + \beta_1)^2]}{(\alpha_1 - \gamma_1)} + \alpha_2(\alpha_1 + \beta_1) \right\} \\
 x_1 &= 2V_0 \left\{ \frac{\alpha_1[\mu - (\alpha_1 + \beta_1)] - \alpha_2(\alpha_1 - \gamma_1)}{\delta H} \right\} \\
 x_2 &= 2V_0 \left\{ \frac{\alpha_1[\delta - (\alpha_1 + \beta_1)\mu + (\alpha_1 + \beta_1)^2] + \alpha_2(\alpha_1 - \gamma_1)(\alpha_1 + \beta_1)}{\delta(\alpha_1 - \gamma_1)H} \right\} \\
 x_3 &= V_0 \left\{ \frac{\alpha_1[\mu - (\alpha_1 + \beta_1)] - \alpha_2(\alpha_1 - \gamma_1)}{\delta} - \frac{2\alpha_1[\mu^2 - (\alpha_1 + \beta_1)\mu - \delta] - 2\alpha_2(\alpha_1 - \gamma_1)\mu}{\delta^2 H^2} \right\} \\
 x_4 &= V_0 \left\{ \frac{\alpha_1[\delta - (\alpha_1 + \beta_1)\mu + (\alpha_1 + \beta_1)^2]}{\delta(\alpha_1 - \gamma_1)} + \frac{\alpha_2(\alpha_1 + \beta_1)}{\delta} + \frac{2\alpha_2[\delta - (\alpha_1 + \beta_1)\mu] - 2\alpha_1\mu[\delta - (\alpha_1 + \beta_1)\mu + (\alpha_1 + \beta_1)^2]}{\delta^2 H^2(\alpha_1 - \gamma_1)} \right\}
 \end{aligned} \quad (21)$$

2 内力计算

2.1 连梁的剪力、墙肢轴力计算

如图 2 所示, 对各墙肢写出竖向平衡方程, 可得:

$$\begin{aligned}
 V_{b,1s} &= \int_0^x \frac{y_1}{2c_1} dx; \quad V_{b,2s} = \int_0^x \frac{y_2}{2c_2} dx; \quad N_{1j} = \sum_{s=j}^n V_{b,1s}; \\
 N_{2j} &= \sum_{s=j}^n V_{b,2s} - \sum_{s=j}^n V_{b,1s}; \quad N_{2j} = \sum_{s=j}^n V_{b,2s}
 \end{aligned} \quad (22)$$

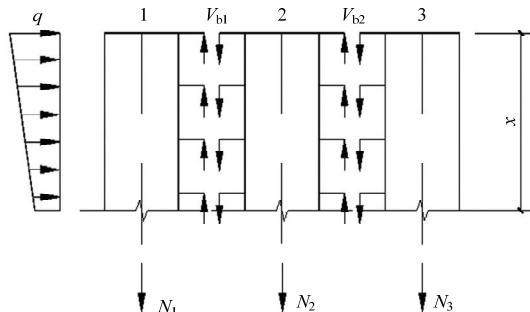


图 2 墙肢轴力计算图

Fig.2 Calculation of axial force of wall limb

2.2 墙肢弯矩计算

各墙肢弯矩可叠合起来, 如图 3 所示。由于曲

率与弯矩的关系及每个墙肢曲率相等的假设^[1]可得到式 (23):

$$\begin{aligned}
 EI_1 \frac{d^2 y_m}{dx^2} &= M_1; \quad EI_2 \frac{d^2 y_m}{dx^2} = M_2; \quad EI_3 \frac{d^2 y_m}{dx^2} = M_3; \\
 E \sum_{i=1}^3 I_i \frac{d^2 y_m}{dx^2} &= \sum_{i=1}^3 M_i \\
 M_i &= \frac{I_i}{\sum_{i=1}^3 I_i} \left(M_p - \int_0^x m(\lambda) d\lambda \right)
 \end{aligned} \quad (23)$$

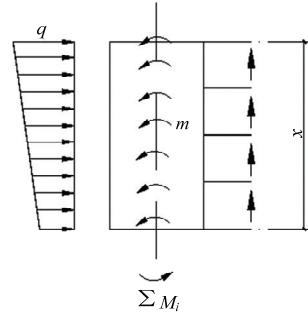


图 3 墙肢弯矩计算图

Fig.3 Calculation diagram of bending moment of wall limb

由式 (23) 及式 (18) ~ 式 (20) 得到不同荷载下墙肢弯矩。

当 $V_p = V_0$ 时:

$$M_i = \frac{I_i}{E \sum_{i=1}^3 I_i} \{ [V_0 - (a_1 + b_1)H]x - (\varphi_1 + \varphi_3)H(e^{k_1 x} + e^{-k_1 x}) + H(\varphi_2 + \varphi_4)(e^{k_2 x} + e^{-k_2 x}) \} \quad (24)$$

当 $V_p = \frac{V_0}{H}x$ 时:

$$\begin{aligned}
 M_i &= \frac{I_i}{E \sum_{i=1}^3 I_i} \left\{ \frac{V_0}{2H} x^2 - \frac{(a_1 + b_1)x^2}{2} - (\varphi_1 + \varphi_3) \left[e^{k_1 x} \left(\frac{e^{-k_1 H}}{k_1} + H \right) - e^{-k_1 x} \left(\frac{e^{k_1 H}}{k_1} - H \right) \right] - \right. \\
 &\quad \left. (\varphi_2 + \varphi_4) \left[e^{k_2 x} \left(\frac{e^{-k_2 H}}{k_2} + H \right) - e^{-k_2 x} \left(\frac{e^{k_2 H}}{k_2} - H \right) \right] \right\}
 \end{aligned} \quad (25)$$

当 $V_p = V_0 \left[1 - \left(1 - \frac{x}{H} \right)^2 \right]$ 时:

$$M_i = \frac{I_i}{E \sum_{i=1}^3 I_i} \left\{ V_0 \left(\frac{x^2}{H} - \frac{x^3}{3H^2} \right) - \left(\frac{x^2}{H} - \frac{x^3}{3H^2} \right) \frac{H(x_1 + x_3)}{2} - \left[x_2 + x_4 - \frac{H(x_1 + x_3)}{2} \right] x - z_1 e^{k_1 x} \left[\frac{1}{k_1} + \frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1(\alpha_1 - \gamma_1)} \right] + z_2 e^{-k_1 x} \left[\frac{1}{k_1} + \frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1(\alpha_1 - \gamma_1)} \right] - z_3 e^{k_2 x} \left[\frac{1}{k_2} + \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2(\alpha_1 - \gamma_1)} \right] + z_4 e^{-k_2 x} \left[\frac{1}{k_2} + \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2(\alpha_1 - \gamma_1)} \right] \right\} \quad (26)$$

相关参数：

$$\begin{aligned} \varphi_1 &= \frac{a_1[k_2^2 - (\alpha_1 + \beta_1)] - b_1(\alpha_1 - \gamma_1)}{(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})}; \quad \varphi_2 = -\frac{a_1[k_1^2 - (\alpha_1 + \beta_1)] - b_1(\alpha_1 - \gamma_1)}{(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} \\ \varphi_3 &= \frac{\left[a_1 \frac{k_2^2 - (\alpha_1 + \beta_1)}{\alpha_1 - \gamma_1} - b_1 \right] [k_1^2 - (\alpha_1 + \beta_1)]}{(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})}; \quad \varphi_4 = -\frac{\left[a_1 \frac{k_1^2 - (\alpha_1 + \beta_1)}{\alpha_1 - \gamma_1} - b_1 \right] [k_2^2 - (\alpha_1 + \beta_1)]}{(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} \\ z_1 &= \frac{e^{-k_1 H} \{x_1[k_2^2 - (\alpha_1 + \beta_1)] - x_2(\alpha_1 - \gamma_1)\}}{k_1(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} + \frac{k_1 \{x_3[k_2^2 - (\alpha_1 + \beta_1)] - x_4(\alpha_1 - \gamma_1)\}}{k_1(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} \\ z_2 &= -\frac{e^{k_1 H} \{x_1[k_2^2 - (\alpha_1 + \beta_1)] - x_2(\alpha_1 - \gamma_1)\}}{k_1(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} + \frac{k_1 \{x_3[k_2^2 - (\alpha_1 + \beta_1)] - x_4(\alpha_1 - \gamma_1)\}}{k_1(k_1^2 - k_2^2)(e^{k_1 H} + e^{-k_1 H})} \\ z_3 &= -\frac{e^{-k_2 H} \{x_1[k_1^2 - (\alpha_1 + \beta_1)] - x_2(\alpha_1 - \gamma_1)\}}{k_2(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} - \frac{k_2 \{x_3[k_1^2 - (\alpha_1 + \beta_1)] - x_4(\alpha_1 - \gamma_1)\}}{k_2(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} \\ z_4 &= \frac{e^{k_2 H} \{x_1[k_1^2 - (\alpha_1 + \beta_1)] - x_2(\alpha_1 - \gamma_1)\}}{k_2(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} - \frac{k_2 \{x_3[k_1^2 - (\alpha_1 + \beta_1)] - x_4(\alpha_1 - \gamma_1)\}}{k_2(k_1^2 - k_2^2)(e^{k_2 H} + e^{-k_2 H})} \end{aligned} \quad (27)$$

2.3 墙肢剪力计算

根据同层水平位移相等，同层节点转角相同的

假设可得到^[1]式 (28)：

$$\text{当 } V_p = V_0 \text{ 时: } V_i = \frac{\tilde{I}_i}{\sum_{i=1}^3 \tilde{I}_i} V_0$$

$$\text{当 } V_p = \frac{V_0}{H} x \text{ 时: } V_i = \frac{\tilde{I}_i}{\sum_{i=1}^3 \tilde{I}_i} \frac{V_0}{H} x$$

$$\text{当 } V_p = V_0 \left[1 - \left(1 - \frac{x}{H} \right)^2 \right] \text{ 时: } V_i = \frac{\tilde{I}_i}{\sum_{i=1}^3 \tilde{I}_i} V_0 \left[1 - \left(1 - \frac{x}{H} \right)^2 \right]$$

$$V_i = \frac{\tilde{I}_i}{\sum_{i=1}^3 \tilde{I}_i} V_p, \text{ 其中: } \tilde{I}_i = \frac{I_i}{1 + \beta_i}, \beta_i = \frac{12\mu EI_i}{h^2 GA_i} \circ \quad (28)$$

$$\begin{aligned} y &= \frac{1}{E \sum_{i=1}^3 I_i} \left\{ V_0 \left(\frac{1}{6} x^3 - \frac{1}{2} H^2 x + \frac{1}{3} H^3 \right) - (\varphi_1 + \varphi_3) \frac{H}{k_1^2} \left[\frac{e^{k_1 x} - e^{-k_1 x} - (e^{k_1 H} - e^{-k_1 H})}{k_1} - (e^{k_1 H} + e^{-k_1 H})(x - H) \right] - \right. \\ &\quad \left. \frac{H(\varphi_2 + \varphi_4)}{k_2^2} \left[\frac{e^{k_2 x} - e^{-k_2 x} - (e^{k_2 H} - e^{-k_2 H})}{k_2} - (e^{k_2 H} + e^{-k_2 H})(x - H) \right] - (a_1 + b_1) H \left(\frac{1}{6} x^3 - \frac{H^2}{2} x + \frac{1}{3} H^3 \right) \right\} - \frac{\mu V_0}{G \sum_{i=1}^3 A_i} (x - H) \end{aligned} \quad (30)$$

3 位移计算

剪力墙的水平位移可由墙肢弯曲变形产生的水平位移 y_m 可由式 (21) 积分求出以及墙肢剪切变形产生的水平位移 y_v ，由剪切变形与墙肢剪力间的下述关系积分求出：

$$\begin{aligned} y &= y_m + y_v = \frac{1}{E \sum_{i=1}^3 I_i} \int \int M_p dx dx - \\ &\quad \frac{1}{E \sum_{i=1}^3 I_i} \int \int \int m(x) dx dx dx - \frac{\mu}{G \sum_{i=1}^3 A_i} \int V_p dx \end{aligned} \quad (29)$$

由式 (26) 积分可得不同荷载下位移表达式及顶部位移。

当 $V_p = V_0$ 时：

当 $x=0$ 时, 顶部位移为:

$$y = \frac{\mu V_0 H}{G \sum_{i=1}^3 A_i} + \frac{1}{E \sum_{i=1}^3 I_i} \left\{ \frac{V_0 H^2}{3} - \frac{(\varphi_1 + \varphi_3) H}{k_1^2} \left[\frac{-(e^{k_1 H} - e^{-k_1 H})}{k_1} + H(e^{k_1 H} + e^{-k_1 H}) \right] - \frac{H(\varphi_2 + \varphi_4)}{k_2^2} \left[\frac{-(e^{k_2 H} - e^{-k_2 H})}{k_2} + H(e^{k_2 H} + e^{-k_2 H}) \right] - \frac{(a_1 + b_1) H^4}{3} \right\} \quad (31)$$

当 $V_p = \frac{V_0}{H} x$ 时:

$$\begin{aligned} y = & -\frac{\mu V_0}{2 G H \sum_{i=1}^3 A_i} (x^2 - H^2) + \frac{1}{E \sum_{i=1}^3 I_i} \left\{ \frac{V_0}{H} \left(\frac{1}{24} x^4 - \frac{H^3}{6} x + \frac{1}{8} H^4 \right) - \left(\frac{\varphi_1 + \varphi_3}{k_1^2} \right) \left[H \frac{(e^{k_1 x} - e^{-k_1 x}) - (e^{k_1 H} - e^{-k_1 H})}{k_1} - H(x-H)(e^{k_1 H} + e^{-k_1 H}) + \frac{e^{k_1(x-H)} + e^{-k_1(x-H)} - 2}{k_1^2} \right] - \left(\frac{\varphi_2 + \varphi_4}{k_2^2} \right) \left[H \frac{(e^{k_2 x} - e^{-k_2 x}) - (e^{k_2 H} - e^{-k_2 H})}{k_2} - H(x-H)(e^{k_2 H} + e^{-k_2 H}) + \frac{e^{k_2(x-H)} + e^{-k_2(x-H)} - 2}{k_2^2} \right] - (a_1 + b_1) \left(\frac{1}{24} x^4 - \frac{H^3}{6} x + \frac{H^4}{8} \right) \right\} \end{aligned} \quad (32)$$

当 $x=0$ 时, 顶部位移为:

$$\begin{aligned} y = & \frac{\mu V_0 H}{2 G \sum_{i=1}^3 A_i} + \frac{1}{E \sum_{i=1}^3 I_i} \left\{ \frac{V_0 H^3}{8} - \frac{(\varphi_1 + \varphi_3)}{k_1^2} \left[-\frac{H(e^{k_1 H} - e^{-k_1 H})}{k_1} + H^2(e^{k_1 H} + e^{-k_1 H}) + \frac{e^{k_1 H} + e^{-k_1 H} - 2}{k_1^2} \right] - \frac{(\varphi_2 + \varphi_4)}{k_2^2} \left[-\frac{H(e^{k_2 H} - e^{-k_2 H})}{k_2} + H^2(e^{k_2 H} + e^{-k_2 H}) + \frac{e^{k_2 H} + e^{-k_2 H} - 2}{k_2^2} \right] - (a_1 + b_1) \frac{H^4}{8} \right\} \end{aligned} \quad (33)$$

当 $V_p = V_0 [1 - (1 - \frac{x}{H})^2]$ 时:

$$\begin{aligned} y = & \frac{\mu V_0}{G \sum_{i=1}^3 A_i} \left(\frac{x^3}{3H^2} - \frac{x^2}{H} + \frac{2H}{3} \right) + \frac{1}{E \sum_{i=1}^3 I_i} \left\{ V_0 \left(-\frac{x^5}{60H^2} + \frac{x^4}{12H} - \frac{H^2}{4} x + \frac{11H^3}{60} \right) + (x_1 + x_3) \left(\frac{x^5}{120H} - \frac{x^4}{24} + \frac{H^3}{8} x - \frac{11H^4}{120} \right) - z_1 \left[\frac{e^{k_1 x} - e^{k_1 H} - k_1 e^{k_1 H} (x-H)}{k_1} \right] \left(\frac{1}{k_1^2} + \frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1^2 (\alpha_1 - \gamma_1)} \right) + z_2 \left[\frac{e^{-k_1 x} - e^{-k_1 H} + k_1 e^{-k_1 H} (x-H)}{k_1} \right] \left(\frac{1}{k_1^2} + \frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1^2 (\alpha_1 - \gamma_1)} \right) - z_3 \left[\frac{e^{k_2 x} - e^{-k_2 H} - k_2 e^{k_2 H} (x-H)}{k_2} \right] \left(\frac{1}{k_2^2} + \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2^2 (\alpha_1 - \gamma_1)} \right) + z_4 \left[\frac{e^{-k_2 x} - e^{-k_2 H} + k_2 e^{-k_2 H} (x-H)}{k_2} \right] \left(\frac{1}{k_2^2} + \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2^2 (\alpha_1 - \gamma_1)} \right) \right\} \end{aligned} \quad (34)$$

当 $x=0$ 时, 顶部位移为:

$$\begin{aligned} y = & \frac{2\mu V_0 H}{3 G \sum_{i=1}^3 A_i} + \frac{1}{E \sum_{i=1}^3 I_i} \left\{ \frac{11V_0 H^3}{60} - \frac{11H^4}{120} (x_1 + x_3) - \frac{H^3}{3} \left[x_2 + x_4 - \frac{H(x_1 + x_3)}{2} \right] - z_1 \frac{1 - e^{k_1 H} + k_1 H e^{k_1 H}}{k_1} \left[\frac{1}{k_1^2} + \frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1^2} \right] + z_2 \frac{1 - e^{-k_1 H} - k_1 H e^{-k_1 H}}{k_1} \left[\frac{1}{k_1^2} + \frac{k_1^2 - (\alpha_1 + \beta_1)}{k_1^2} \right] - z_3 \frac{1 - e^{k_2 H} + k_2 H e^{k_2 H}}{k_2} \left[\frac{1}{k_2^2} + \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2^2} \right] + z_4 \frac{1 - e^{-k_2 H} - k_2 H e^{-k_2 H}}{k_2} \left[\frac{1}{k_2^2} + \frac{k_2^2 - (\alpha_1 + \beta_1)}{k_2^2} \right] \right\} \end{aligned} \quad (35)$$

4 算例分析

图4所示11层3肢剪力墙，弹性模量 $E=2.6 \times 10^7 \text{ kN/m}^2$ ，剪切模量为 G ，墙厚0.2m，设剪力墙的截面面积为 A_i ，惯性矩为 I_i ，折算惯性矩为 \tilde{I}_i ，连梁的计算跨度为 a_i ，惯性矩为 I_{bi} ，折算惯性矩为 \tilde{I}_{bi} ，求其内力和位移。

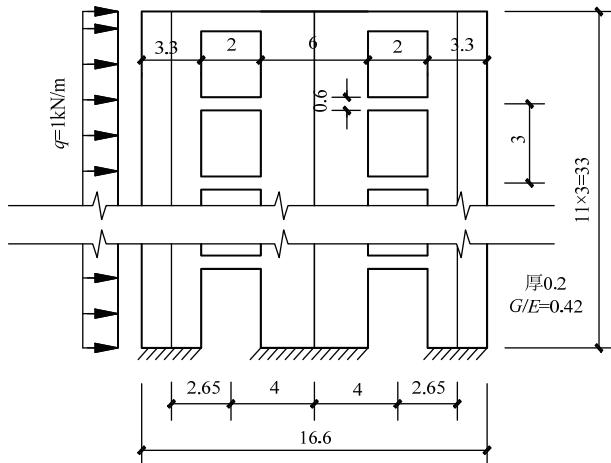


图4 11层3肢剪力墙(单位:m)

Fig.4 11 Story 3 leg shear wall (unit: m)

墙肢连梁相关参数计算的系数见表1。

表1 墙肢连梁参数

Table 1 Parameters of beam coupling

参数名称	参数数值	求和Σ
墙肢截面面积 A_s	0.66	1.2
墙肢惯性矩 I_i	0.59895	3.6
墙肢分配系数	0.12484	0.75032
墙肢折算惯性矩 \tilde{I}_i	0.13438	0.28966
墙肢折算惯性矩分配系数	0.24064	0.51871
连梁惯性矩 I_{bi}		0.0036
连梁折算惯性矩 \tilde{I}_{bi}		0.003

4.1 计算步骤

(1) 根据基本数据通过式(11)、式(13)、式(23)、式(29)计算有关参数代入式(21)中，这时约束弯矩 y_1 ， y_2 已求出。

(2) 根据约束弯矩 y_1 ， y_2 ，由式(24)~式(30)计算墙肢、连梁内力。

(3) 由式(35)计算水平位移，综合参数计算见表2。

根据相关系数代入式(21)得到连梁约束弯矩

$$y_1 = -4.51152 \times 10^{-3} \times (33.06962e^{0.14027x} - 697.037675e^{-0.14027x}) + 1.2561 \times 10^{-6} \times (35.81057e^{0.05602x} - 83.37563e^{-0.05602x}) + 0.462026x$$

$$y_2 = -4.5113 \times 10^{-3} \times (33.06962e^{0.14027x} - 697.037675e^{-0.14027x}) - 1.2561 \times 10^{-6} \times (35.81057e^{0.05602x} - 83.37563e^{-0.05602x}) + 0.46202x$$

在徐彬等^[14]的奇异函数解法可以处理离散问题，直接确定连梁弯矩，但本文用的连梁连续假设，故把连梁约束弯矩从0~1.5m积分作为第11层弯矩，1.5~4.5m作为第10层弯矩以此类推计算每层约束弯矩由于约束弯矩相差不大，连梁剪力及杆端弯矩按左侧计算，内力计算结果列于表3。表3中 $m_{1,j}$ $m_{2,j}$ 为两连梁约束弯矩； $V_{b,j}$ 为连梁剪力； $M_{b,j}$ 为连梁杆端弯矩； M_1 ， M_2 ， M_3 为墙肢弯矩； V_1 ， V_2 ， V_3 为墙肢剪力。

表2 综合参数

Table 2 Comprehensive parameters

参数名称	参数数值
c_i^2	11.0556
α_i	9.0905×10^{-3}
β_i	0.0023163
γ_i	0.00082190
δ	6.1745×10^{-5}
μ	0.0228136
k_i^2	0.019675
b_i	0.462026
φ_i	-4.51152×10^{-3} 1.2561×10^{-6} -4.5113×10^{-3} -1.2561×10^{-6}

文献[1]经典例题计算结果如表4所示。

为验证其正确性由Midas建模，剪力墙、连梁用的是梁单元，之间连接为刚性连接，支座为固定端支座，左侧加1kN/m的均布荷载，采用C20混凝土，弹性模量为 2.55×10^7 (kN/m²) 其他截面尺寸和题目相同。得到的连梁弯矩图如图5所示。

对比两方法连梁约束弯矩作图像对比如图6所示。

由图像可知解析方法求出的连梁弯矩大于传统方法求出的弯矩，平均大10%左右，在6~8层解析法结果几乎与数值模拟结果一样，其他楼层与模型相差不大，总的来说解析法更接近数值模拟结果。

(4) 位移计算

由式(32)得，当 $x=0$ 时顶部位移：

表3 内力计算结果
Table 3 Calculation results of internal forces

层	x	$m_{1,j}$	$m_{2,j}$	$\sum_{s=j}^n m_s$	M_p	$V_{b,j}$ (kN)	$M_{b,j}$
11	0~1.5	4.5245	4.5245	9.049	1.125	0.6804	0.6804
10	1.5~4.5	9.7105	9.7106	28.4702	10.125	1.46023	1.46023
9	4.5~7.5	11.3661	11.3664	51.2026	28.125	1.70919	1.70919
8	7.5~10.5	13.5699	13.5703	78.3435	55.125	2.04069	2.04069
7	10.5~13.5	15.9705	15.971	110.2843	91.125	2.40147	2.40147
6	13.5~16.5	18.2523	18.253	147.5505	136.125	2.74471	2.74471
5	16.5~19.5	20.0779	20.0787	186.9462	190.125	3.01923	3.01923
4	19.5~22.5	21.0284	21.0294	229.004	253.125	3.16217	3.16217
3	22.5~25.5	20.5272	20.5284	270.0596	325.125	3.08680	3.08680
2	25.5~28.5	17.737	17.7385	305.5351	406.125	2.66722	2.66722
1	28.5~31.5	11.4092	11.4111	328.3554	496.125	1.71567	1.71567
0	31.5~33	1.7344	1.7357	331.8255	544.5	0.26081	0.26081

层	x	$M_p - \sum_{s=j}^n m_s$	$M_1=M_3$ (kN·N)	M_2 (kN·N)	$V_1=V_3$ (kN)	V_2 (kN)	$N_1=N_3$ (kN)	N_2 (kN)
11	0~1.5	-7.924	-0.989	-5.946	0	0	0.6804	0
10	1.5~4.5	-18.345	-2.2901	-13.765	0.722	1.556	1.4602	0
9	4.5~7.5	-23.078	-2.881	-17.316	1.444	3.112	1.7091	0
8	7.5~10.5	-23.219	-2.899	-17.422	2.166	4.668	2.0406	0
7	10.5~13.5	-19.159	-2.392	-14.375	2.887	6.224	2.4014	0
6	13.5~16.5	-11.426	-1.426	-8.573	3.609	7.780	2.7447	0
5	16.5~19.5	3.179	0.397	2.385	4.331	9.336	3.0192	0
4	19.5~22.5	24.121	3.011	18.098	5.053	10.892	3.1621	0
3	22.5~25.5	55.0654	6.874	41.317	5.769	12.437	3.0868	0
2	25.5~28.5	100.590	12.558	75.475	6.497	14.004	2.6672	0
1	28.5~31.5	167.770	20.944	125.881	7.299	15.733	1.7156	0
0	31.5~33	212.675	26.550	159.574	7.941	17.117	0.2608	0

表4 内力计算结果
Table 4 Calculation results of internal forces

层	x	m_j	$\sum_{s=j}^n m_s$	M_p	$V_{bl,j}$ (kN)	$M_{b,j}$	$M_p - \sum_{s=j}^n m_s$	$M_1=M_3$ (kN·N)	M_2 (kN·N)	$V_1=V_3$ (kN)	V_2 (kN)	$N_1=N_3$ (kN)
11	0	7.405	7.405	0	0.557	0.557	-7.405	-0.924	-5.556	0	0	0.557
10	3	16.157	23.562	4.5	1.215	1.215	-19.063	-2.380	-14.303	0.722	1.556	1.772
9	6	19.246	42.808	18	1.447	1.447	-24.812	-3.098	-18.617	1.444	3.112	3.219
8	9	23.364	66.172	40.5	1.757	1.757	-25.680	-3.206	-19.268	2.166	4.668	4.976
7	12	27.958	93.830	72	2.102	2.102	-21.847	-2.727	-16.392	2.887	6.224	7.078
6	15	32.314	126.144	112.5	2.430	2.430	-13.667	-1.706	-10.255	3.609	7.780	9.508
5	18	35.957	162.101	162	2.704	2.704	-0.134	-0.017	-0.101	4.331	9.336	12.212
4	21	38.412	200.513	220.5	2.888	2.888	19.944	2.490	14.964	5.053	10.892	15.100
3	24	37.858	238.371	288	2.846	2.846	49.572	6.189	37.195	5.769	12.437	17.946
2	27	33.660	272.031	364.5	2.531	2.531	92.397	11.535	69.327	6.497	14.004	20.477
1	30	22.176	294.207	450	1.667	1.667	155.702	19.438	116.826	7.299	15.733	22.144
0	33	0	294.207	544.5	0	0	250.293	31.247	187.800	7.941	17.117	22.144

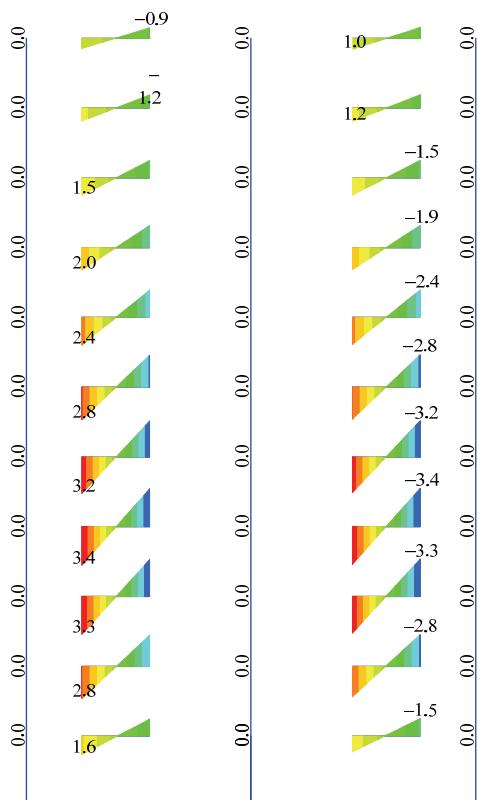


图 5 连梁弯矩图

Fig.5 Bending moment diagram of coupling beam

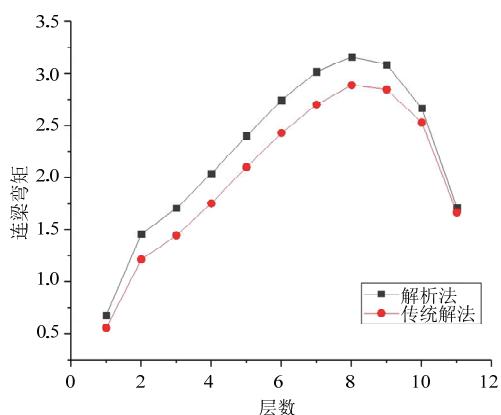


图 6 连梁弯矩对比图

Fig.6 Bending moment comparison diagram of coupling beam

$$\begin{aligned}
 y = & \frac{1.2 \times 33 \times 33}{2 \times 0.42 \times 2.6 \times 10^7 \times 2.52} + \frac{1}{2.6 \times 10^7 \times 4.7979} \times \\
 & \left\{ \frac{33^4}{8} - (-0.45859) \times \left[\frac{-33 \times (e^{4.628844} - e^{-4.628844})}{0.140268} + \right. \right. \\
 & 33^2 \times (e^{4.628844} + e^{-4.628844}) + \frac{e^{4.628844} + e^{-4.628844} - 2}{0.019675} \left. \left. \right] - \right. \\
 & 0.924052 \times \frac{33^4}{8} \left. \right\} = 0.0004304298(m)
 \end{aligned}$$

5 结论与展望

(1) 虽然解析解推导过程复杂，但其结果简单明了，适用于大量的实际工程。其结果与软件模拟十分接近，更能反应结构力学特性，在结构高度较高时优势明显，与传统解法相比，其结果应用简单，不需反复查表，精度高，充分发挥了解析式优势。

(2) 传统方法为近似解，是微分方程叠加为一个方程后分配，其近似程度不可定量计算，本文是通过换元降阶的方法将二阶微分方程组拓展成一阶微分方程组，该方程组的解与原方程组的解是等价的，其解为基解矩阵的第一行，在数学中已经有严格证明，在满足基本假定情况下，该解为解析解，更接近实际情况。

(3) 当墙肢的相关参数沿高度变化时，即系数矩阵含有未知数，同样可解，该方法同样适用，后续可以进一步研究。

(4) 该方法可为以后四肢等多肢墙解析解法提供思路，为其后多肢剪力墙及其相关结构的解法提供理论指导。

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